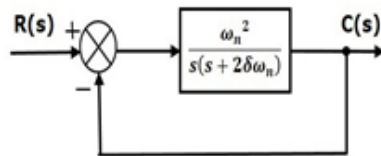




Time Response of Second Order Control System

The order of a control system is determined by the power of 's' in the denominator of its transfer function. If the power of s in the denominator of the transfer function of a control system is 2, then the system is said to be **second order control system**. The general expression of the transfer function of a second order control system is given as



$$G(S) = \frac{\omega_n^2}{s(s+2\xi\omega_n)}$$

$$H(S) = 1$$

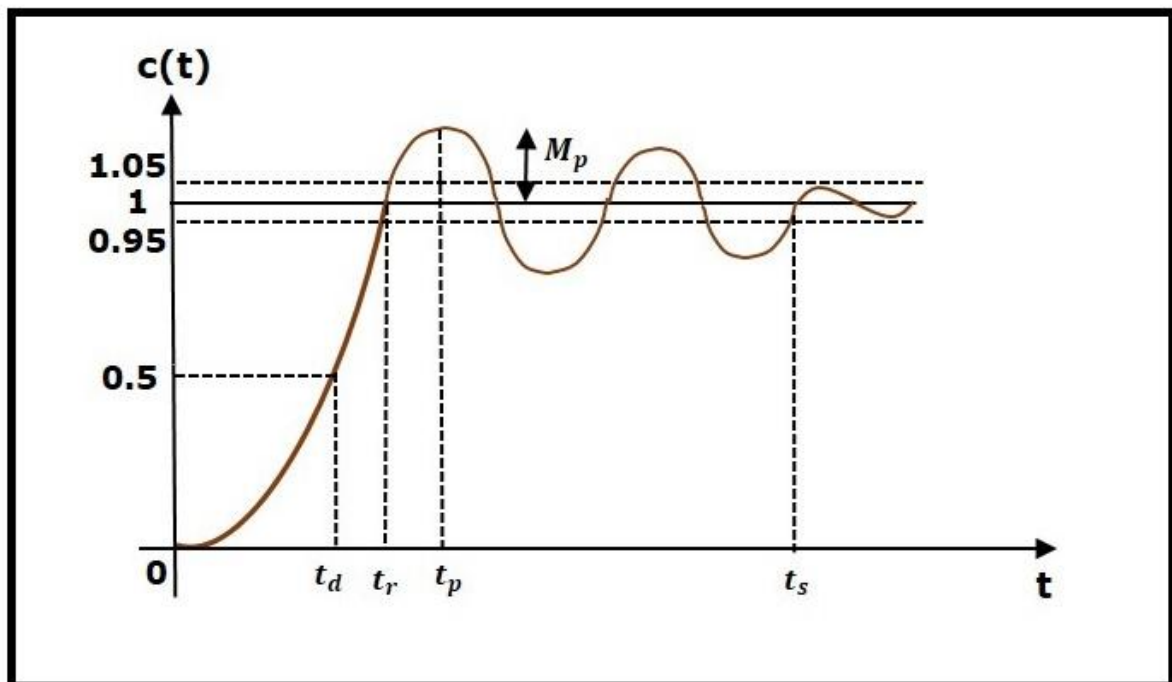
$$\frac{C(s)}{R(s)} = \frac{G(S)}{1 + G(S)H(S)}$$

$$\frac{C(S)}{R(S)} = \frac{\omega_n^2/s(s + 2\xi\omega_n)}{1 + \frac{\omega_n^2}{s(s + 2\xi\omega_n)}}$$

$$\boxed{\frac{C(S)}{R(S)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}} \text{eq.1}$$

Transient response specification of second order system

The performance of the control system are expressed in terms of transient response to a unit step input because it is easy to generate initial condition basically are zero.

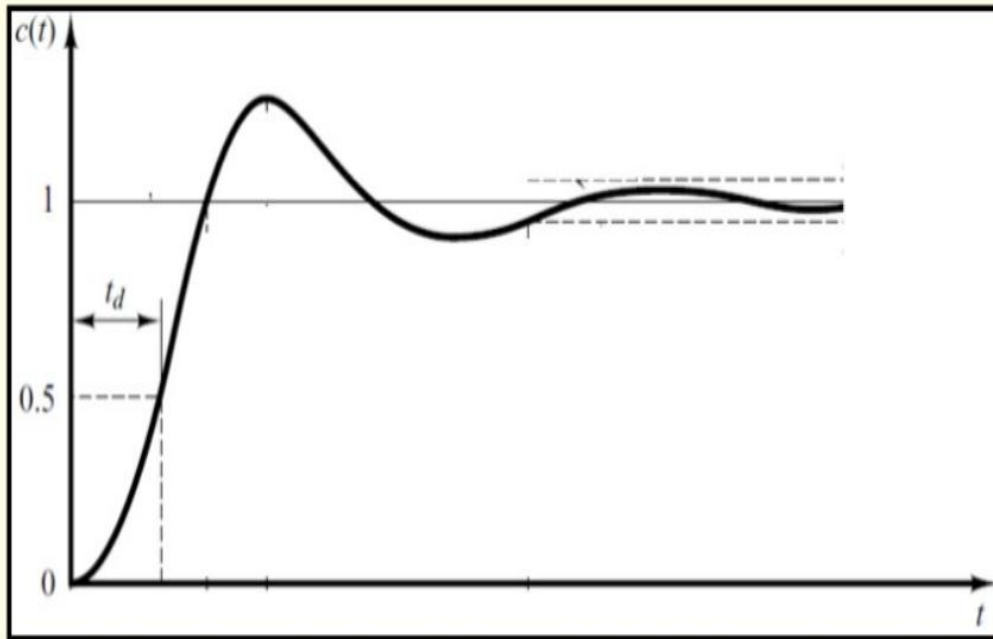


Following are the common transient response characteristics:

1. Delay Time.

The time required for the response to reach **50%** of the final value in the first time is called the delay time.

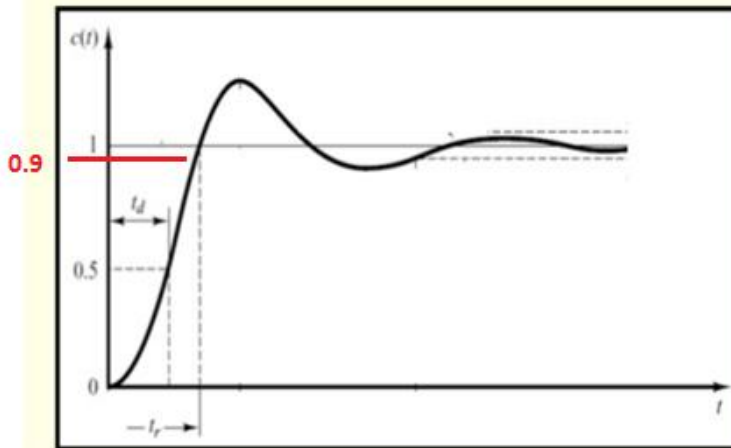
- **Delay-Time (T_d)**: The delay (T_d) time is the time required for the response to reach half the final value the very first time.



2. Rise Time.

The time required for response to rising from 10% to 90% of final value, for an overdamped system and 0 to 100% for an underdamped system is called the rise time of the system.

- 10% to 90% of its final value, → over damped systems
- 5% to 95% of its final value, → Critical damped systems
- or 0% to 100% of its final value. → under damped systems



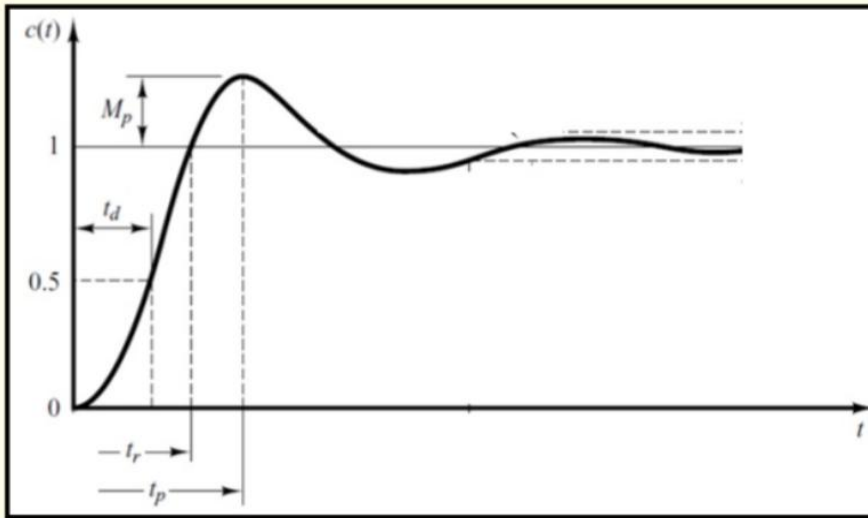
$$t_r = \frac{\pi - \theta}{\omega_d}$$

$$\theta = \tan^{-1} \left(\frac{\omega_n \sqrt{1 - \zeta^2}}{\zeta \omega_n} \right)$$

$$t_r = \frac{\pi - \tan^{-1} \frac{\sqrt{1 - \xi^2}}{\xi}}{\omega_n \sqrt{1 - \xi^2}}$$

3. Peak Time.

- **Peak Time (T_p):** The peak time is the time required for the response to reach the first (maximum) peak of the overshoot.

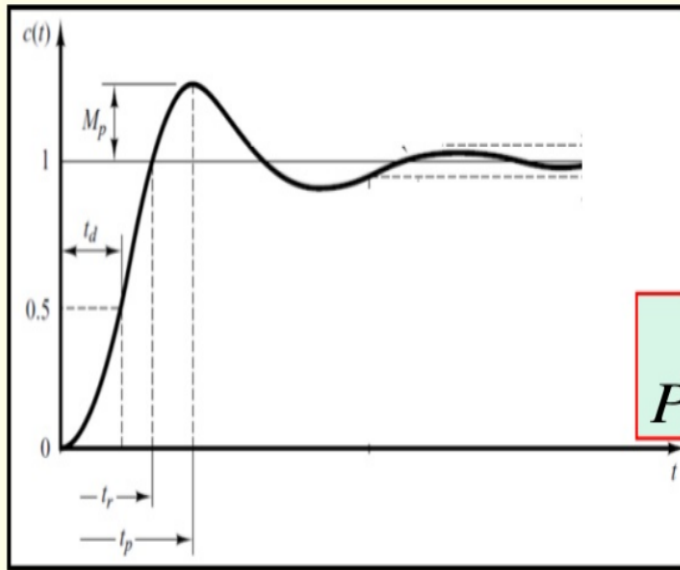


$$t_p = \frac{\pi}{\omega_d}$$

$$t_p = \frac{\pi}{\omega_n \sqrt{1 - \xi^2}}$$

4. Maximum Peak.

The difference between the peak of 1st time and steady output is called the maximum overshoot. It is defined by



$$MP = e^{-\left(\frac{\zeta}{\sqrt{1-\zeta^2}}\right)\pi}$$

$$P.O = \frac{M_p}{\text{final value}} \times 100\%$$

$$P.O = e^{-\frac{\zeta\pi}{\sqrt{1-\zeta^2}}} \times 100\%$$

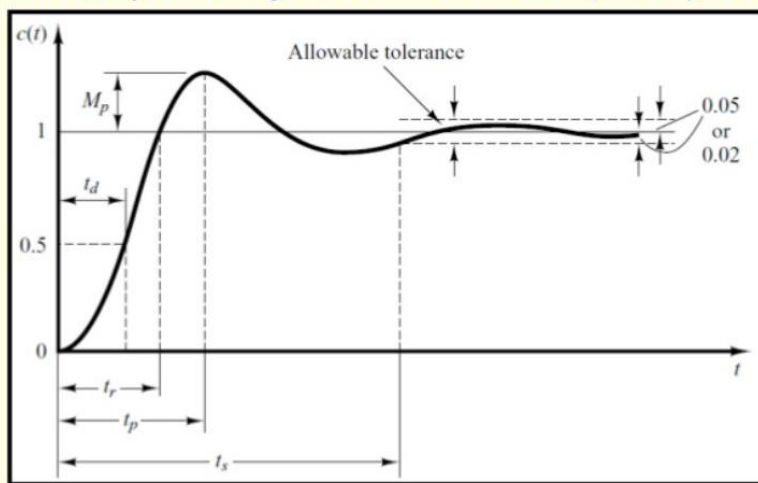
$$\text{Maximum percent overshoot} = \frac{C(t_p) - C(\infty)}{C(\infty)}$$

$$\%M_p = e^{\frac{\pi\zeta}{\sqrt{1-\zeta^2}}} \times 100$$

5. Settling Time.

The time that is required for the response to reach and stay within the specified range (2% to 5%) of its final value is called the settling time.

- The settling time (T_s): is the time required for the response curve to reach and stay within a range about the final value of size specified by absolute percentage of the final value (usually 2% or 5%).



Settling Time (2%)

$$t_s = \frac{4}{\zeta\omega_n}$$

Settling Time (5%)

$$t_s = \frac{3}{\zeta\omega_n}$$

$$t_s = \frac{4}{\zeta\omega_n}$$

6. Steady State error.

The difference between actual output and desired output as time 't' tends to infinity is called the steady state error of the system.

$$e_{ss} = \lim_{t \rightarrow \infty} [r(t) - C(t)]$$

□ $\zeta \rightarrow$ **damping ratio** of the second order system, which is a measure of the degree of resistance to change in the system output.

□ $\omega_n \rightarrow$ **un-damped natural frequency** of the second order system, which is the frequency of oscillation of the system without damping

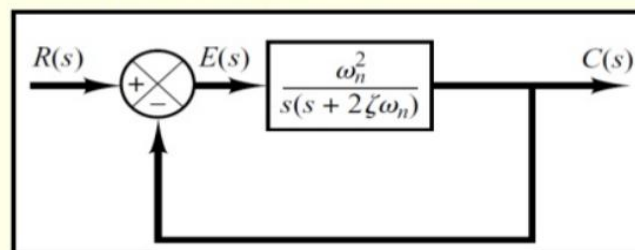
□ According the value of ζ , a second-order system can be set into one of the four categories:

- Case 1: Over damped response ($\xi > 1$)
- Case 2: Critically damped response ($\xi = 1$)
- Case 3: Under damped response ($0 < \xi < 1$)
- Case 4: No damped response ($\xi = 0$)

Example 1

□ Consider the system shown in following figure, where damping ratio is **0.6** and natural undamped frequency is **5 rad/sec**.

Obtain the rise time t_r , peak time t_p , maximum overshoot M_p , and settling time 2% and 5% criterion t_s when the system is subjected to a unit-step input.





□ Solution:

$$\xi = 0.6 \quad \omega_n = 5 \text{ rad/sec}$$

1. Rise Time:

$$t_r = \frac{\pi - \theta}{\omega_d}$$

$$t_r = \frac{3.141 - \theta}{\omega_n \sqrt{1 - \zeta^2}}$$



$$\theta = \tan^{-1}\left(\frac{\omega_n \sqrt{1 - \zeta^2}}{\zeta \omega_n}\right) = 0.93 \text{ rad}$$

$$t_r = \frac{3.141 - 0.93}{5\sqrt{1 - 0.6^2}} = 0.55s$$

2. Peak Time:

$$t_p = \frac{\pi}{\omega_d}$$



$$t_p = \frac{3.141}{4} = 0.785s$$



3. Settling Time (2%):

$$t_s = \frac{4}{\zeta\omega_n}$$



$$t_s = \frac{4}{0.6 \times 5} = 1.33s$$

Settling Time (5%):

$$t_s = \frac{3}{\zeta\omega_n}$$



$$t_s = \frac{3}{0.6 \times 5} = 1s$$

4. Maximum Overshoot:

$$M_p = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}} = e^{-\frac{3.14 \times 0.6}{\sqrt{1-0.6^2}}} = 0.095$$

$$P.O = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}} \times 100 = 9.5\%$$